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Stress Tensor

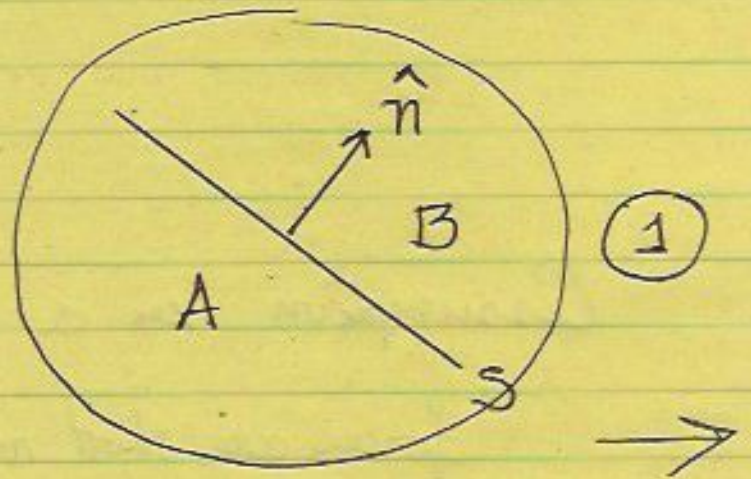
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|-------------------------------|---|
| Definition: forces in a solid | 1 |
| Forces are given by a tensor | 2 |
| 2-D picture | 3 |
| Stress tensor is symmetric | 3 |
| Transformation properties | 5 |
| Principal axes | 6 |

Note Use the matrix notation
in p 5 from the word go

{ Pressure / shear forces decomposition
of the stress tensor } 7

Stress Tensor

Let \vec{x} be an inertial, cartesian, coordinate system, and consider some substance/material under stress



Define, for any element of surface $d\vec{S}$ with unit normal \hat{n} (see figure)

$$\vec{F} = \vec{F}(\hat{n}, \vec{x}) = \left. \begin{array}{l} \text{force per unit} \\ \text{area across the surface} \\ \text{by ~~material~~ side B on side A} \end{array} \right\} \textcircled{2}$$

Since \vec{F} = linear momentum flux, by the same argument as in the problem "why is the flux a vector"

\vec{F} is given by a tensor Namely

$$\vec{F} = \hat{n} \cdot \tau \quad \text{or} \quad F_n = \hat{n}_m \tau_{mn} \quad (3)$$

where $\tau = \tau(\vec{x})$ is the stress tensor

Notation (4)

a) We use the repeated index summation convention

b) The dot denotes tensor contraction,

so that $\hat{n} \cdot \tau = \hat{n}_j \tau_{ji}$ and

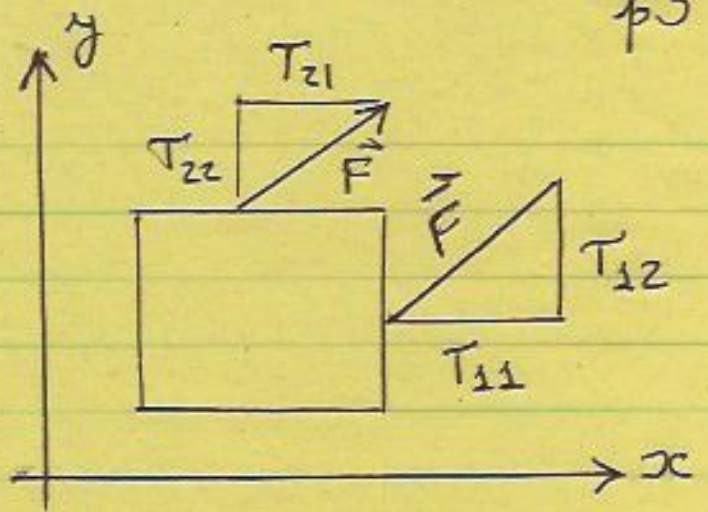
$$\tau \cdot \hat{n} = \tau_{ji} \hat{n}_i$$

c) ∇ is the vector $\frac{\partial}{\partial x_j}$

d) $\text{div } \tau = (\tau_{ji})_{x_j}$

e) τ^+ is the transpose of τ

Two-D picture
illustrating the
meaning of τ



τ is a symmetric tensor (5)

$$\tau_{ij} = \tau_{ji}$$

Proof Consider a sphere of radius R centered at some arbitrary point \vec{x}_0 . Denote the sphere by Ω and its boundary by $\partial\Omega$. The torque on the sphere is then

$$\begin{aligned} \vec{T} &= \int_{\partial\Omega} (\hat{n} \cdot \tau) \times (\vec{x} - \vec{x}_0) dS = \\ &= \int_{\Omega} \operatorname{div} [\tau \times (\vec{x} - \vec{x}_0)] dV \end{aligned}$$

where

$$\operatorname{div} \left[\mathbf{T} \times \underbrace{(\vec{x} - \vec{x}_0)}_{\vec{z}} \right] = (\operatorname{div} \mathbf{T}) \times (\vec{x} - \vec{x}_0) + \left\{ T_{ij} \epsilon_{lji} \right\}$$

$$\text{Since } (\mathbf{T} \times \vec{z})_{il} = T_{ij} \epsilon_{ljm} z_m$$

However, in order to avoid unbounded angular accelerations[†], $\vec{\mathbb{T}}$ must behave like $O(R^4)$ as $R \rightarrow \infty$.

$$\text{Hence } T_{ij} \epsilon_{lji} = 0 \quad \forall l \Rightarrow T_{ij} = T_{ji}$$

† If \vec{a} is the acceleration and ρ the mass density, then

$$\int_{\Omega} \rho \vec{a} \times (\vec{x} - \vec{x}_0) dV = \vec{\mathbb{T}}$$

ignoring body forces and assuming Ω is not changing

Transformation Properties

Let the two cartesian coordinates \vec{x}_a & \vec{x}_b be related by the orthogonal transf. T. Namely

$$\boxed{\vec{x}_b = T \vec{x}_a, \text{ Then } T_b = T T_a T^+} \quad (6)$$

Proof Use matrix notation, with vectors being column vectors and T the matrix $\{T_{ij}\}$. Then (3) \iff

$$\boxed{\vec{F} = T^+ \hat{n} = T \hat{n} \quad (7)}$$

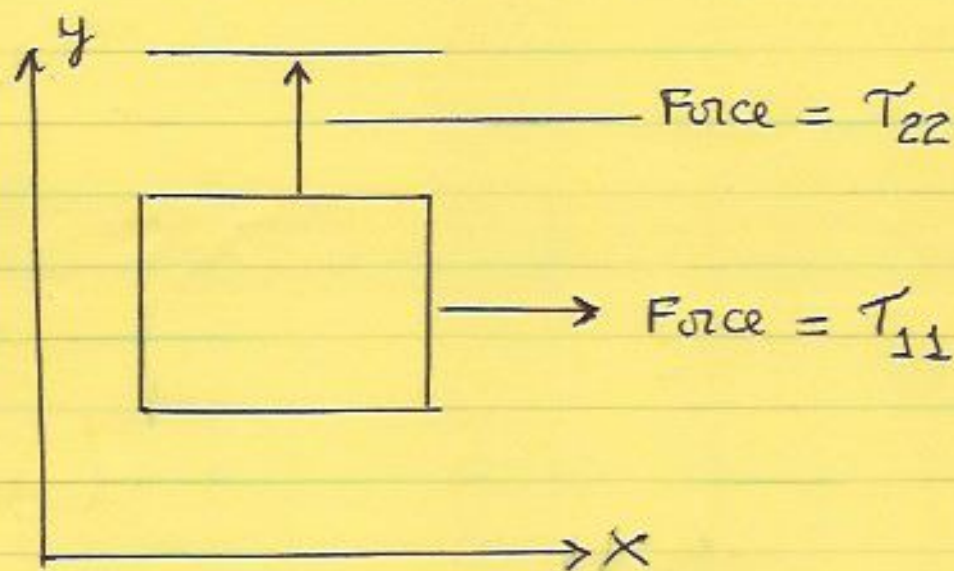
$T = T^+$ from (5)

Now $\vec{F}_b = T \vec{F}_a, \hat{n}_b = T \hat{n}_a \implies$

$$\vec{F}_b = T \vec{F}_a = T T_a \hat{n}_a = \underline{T T_a T^+} \hat{n}_b$$

Principal Axes & physical meaning

At each point a rotation can be applied that makes τ diagonal. The coordinate axis where this happens are the principal axes. Then



(8)

{ Squeeze and pull. No shear
along sides of elementary cube of
sides dx

Pressure - shear decomposition of stress tensor

$$\text{Let } p = -\frac{1}{3} \text{Trace } T = T_{nn}$$

$$\text{Then } T = -p I + T^* \quad (3)$$

Where T^* is symmetric with zero trace (hence the sum of the 3 principal forces by T^* is zero)

p is the pressure. If $T^* = 0$ then the stress is normal to any force, and independent of direction.

The response to pressure only ~~is~~ by an isotropic material will be a volume change.

In elastic, isotropic, materials p is caused by volume changes (same as for fluids) \longrightarrow

On the other hand, if $p = 0$, the response of an isotropic material will be shape changes (with the same volume) In fluids T^* is produced by shear (viscous forces) while in elastic media it results from shear deformations